

# Seismic Response Control for Parallel Building Structures Using MR Damper



Shinya Soma

(Nihon Univ.)

Takashi Tanaka

(Nihon Univ.)

Toru Watanabe

(Nihon Univ.)

Kazuto Seto

(Nihon Univ.)

## Background (1/2)

Buildings are getting higher.



Natural frequencies are getting lower.



**Disturbance such as winds,  
earthquakes, etc**

Structural vibrations become a serious problem.

**Reduction of the vibrations is required.**



**Shinjyuku, Tokyo**



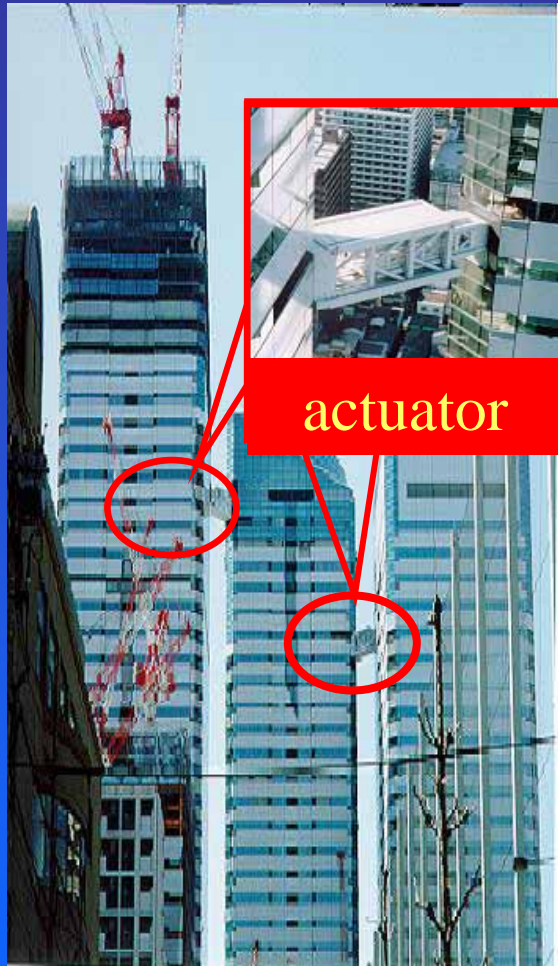
**Great Hanshin-Awaji  
Earthquake**

## Background (2/2)

- **Connected buildings control method is available for vibration control of multiple buildings, and obtain large control force at low frequency.**
- **Semi-active control systems have large control force with lower energy than energy required in active control system, and inherent stability.**
- **Recently, a MR damper is introduced as a semi-active device.**
- **They can be controllable for large vibrations like earthquakes.**

# Connected Buildings Control (CBC) Method

A vibration control mechanism using the interaction of the multiple buildings by installing the control equipment between parallel buildings.



**Triton tower**

## merit

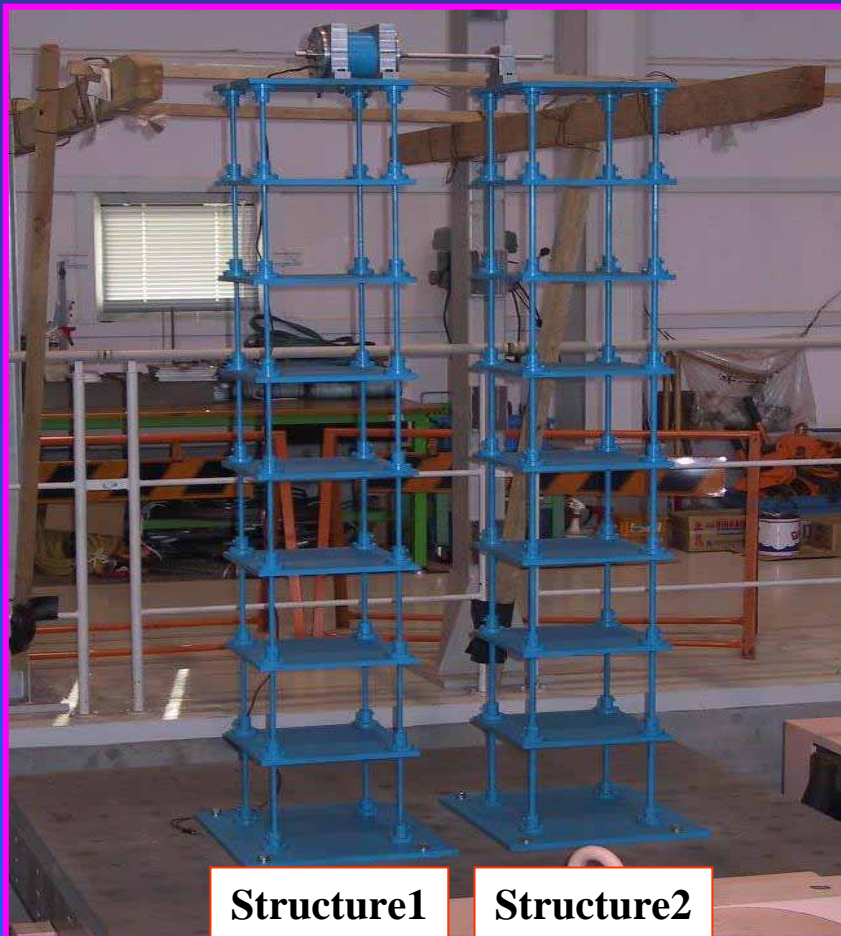
- A large control force can be obtained at low frequency vibrations
- No need for additional mass like a TMD.
- Reduce a number of vibration control devices.
- Useful space inside the control equipment used as a passageway to connect buildings.

# Purpose

To propose a design method of a semi-active control system with a suitable MR damper

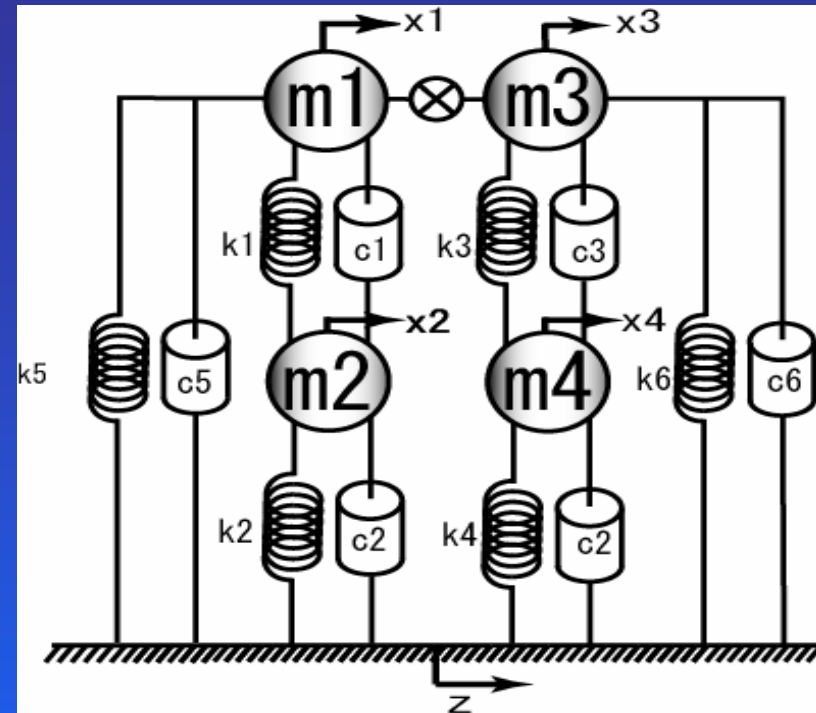
- **To verify the effectiveness of semi-active vibration control for buildings connected with the MR damper by the simulation and experiment.**
- **To propose a design method of a feedback gain using a genetic algorithm for MR damper characterized by non-linearity.**

# Controlled Objects and Model



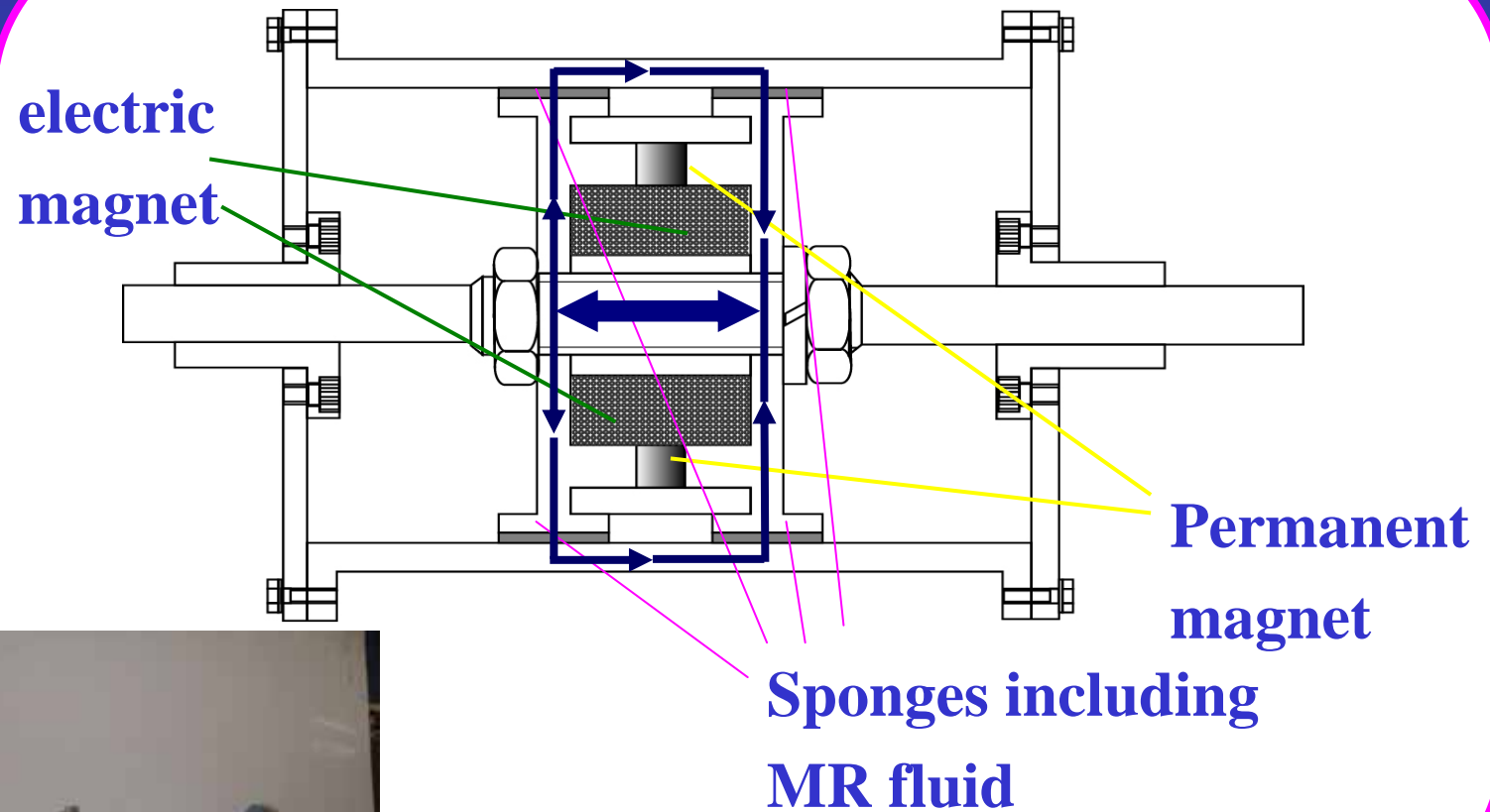
**Structure1**      **Structure2**

Natural frequency	1st mode	2nd mode
Structure1	5.55	16.69
structure2	7.96	23.9



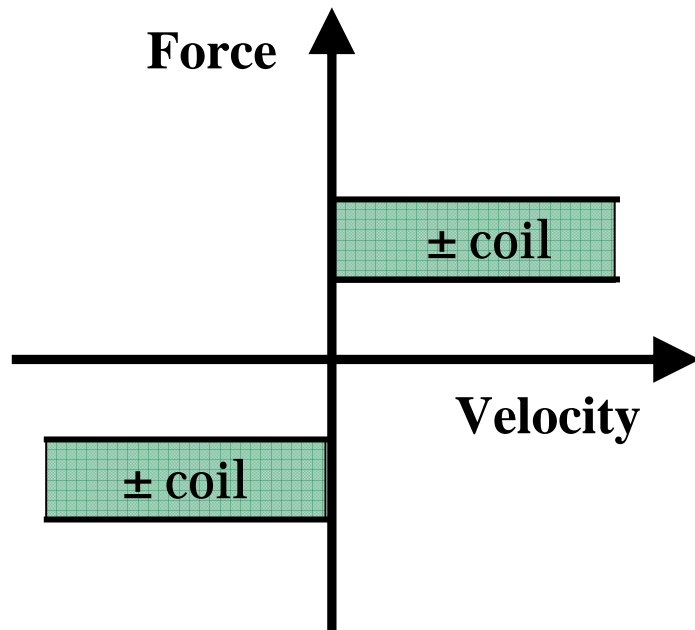
Mass[kg]	Stiffnes[N/m]	Damping[Ns/m]
$m_1=35.58$	$k_1=1.89 \times 10^5$	$c_1=10$
$m_2=43.61$	$k_2=2.81 \times 10^5$	$c_2=47$
	$k_{11}=-5.62 \times 10^4$	$c_{11}=4$
$m_3=27.61$	$K_3=3.08 \times 10^5$	$c_3=6$
$m_4=34.75$	$k_4=3.53 \times 10^5$	$c_4=60$
	$k_{22}=-6.91 \times 10^4$	$c_{22}=10$

# Sponge type Hybrid MR damper

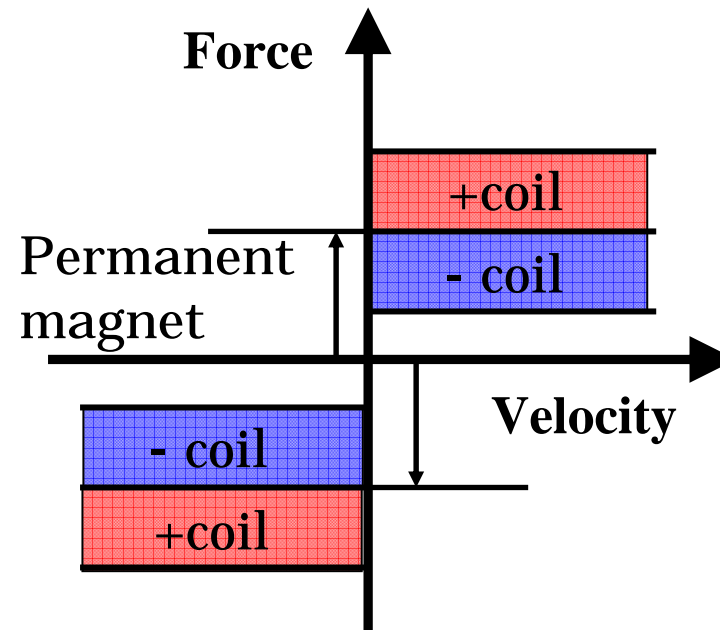


# Range of Variable Damping Force

Compare the range of variable damping force

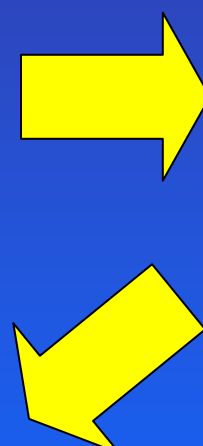
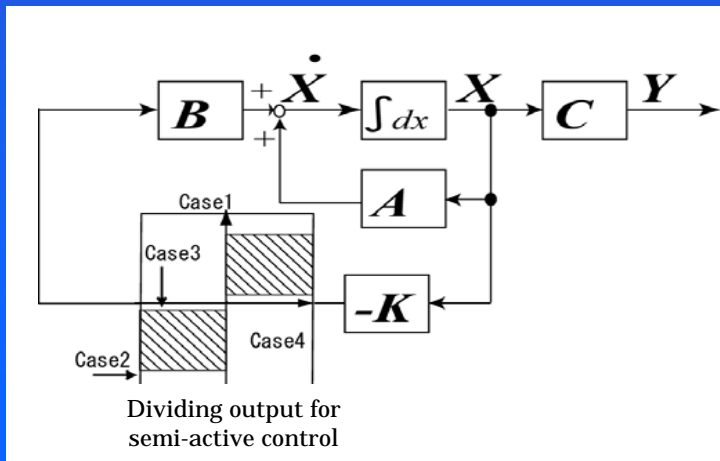
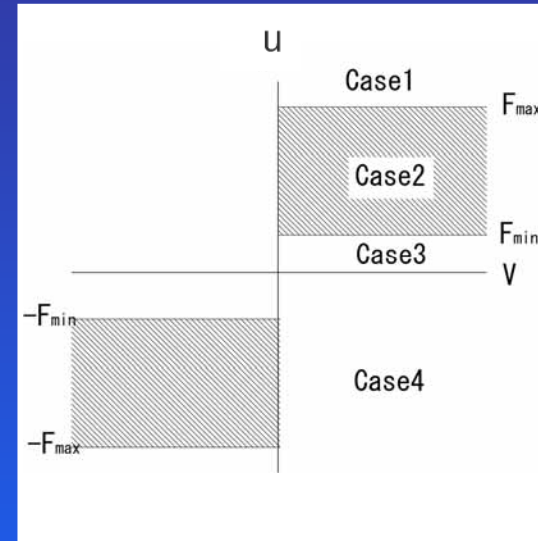
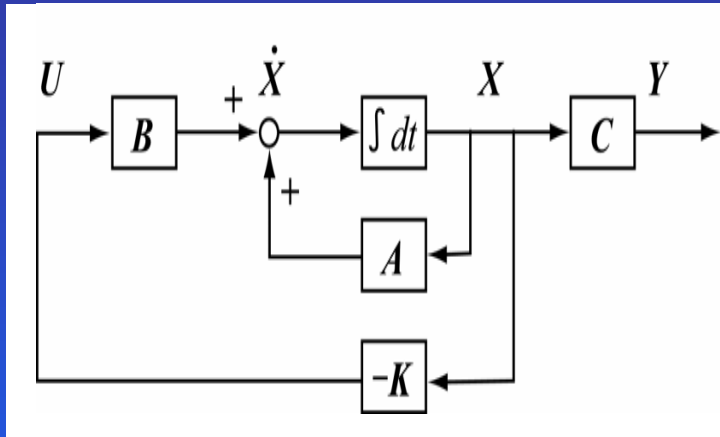


Only a electric magnet



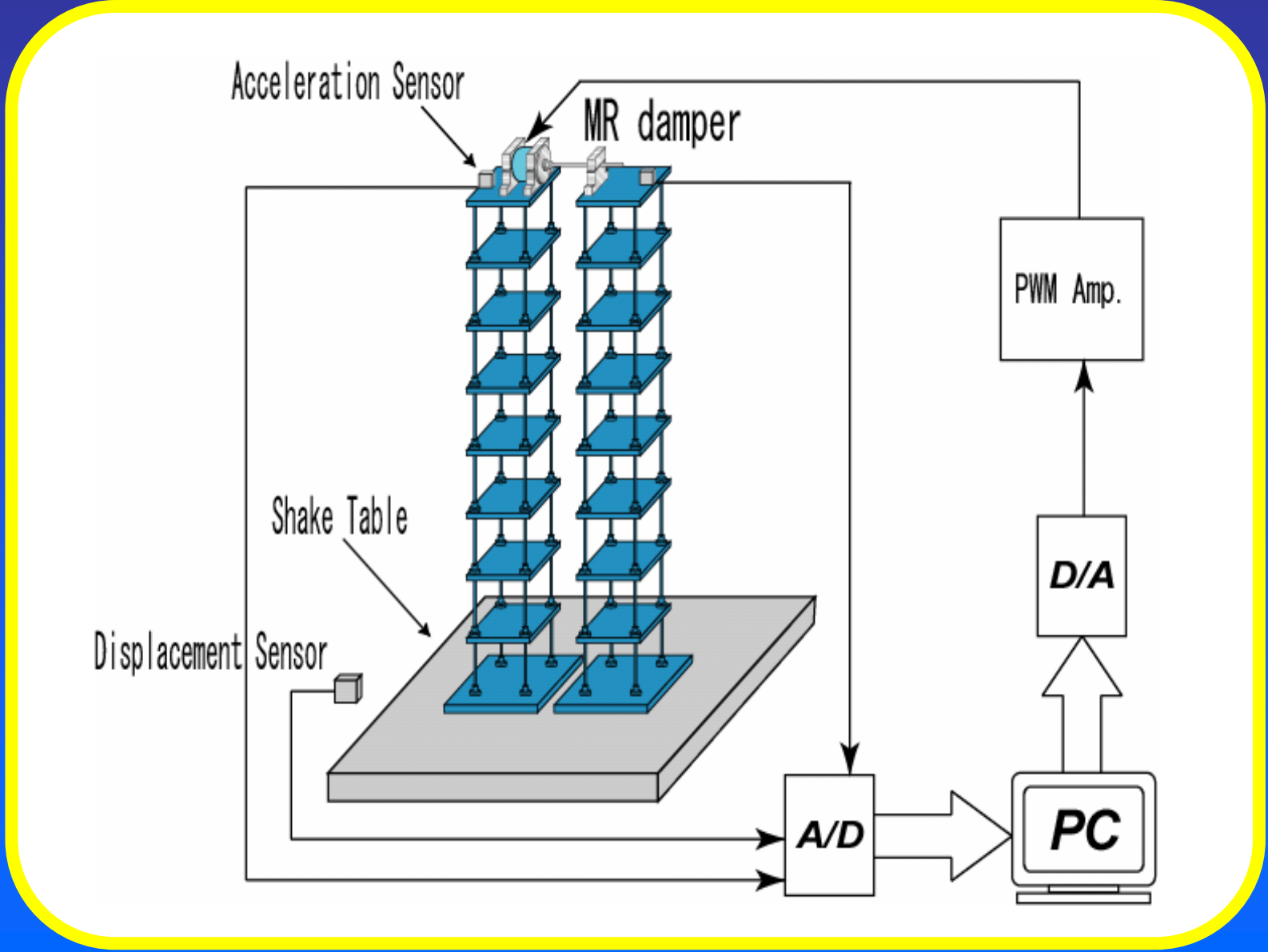
Combine electric and permanent magnets

# Semi-active Control



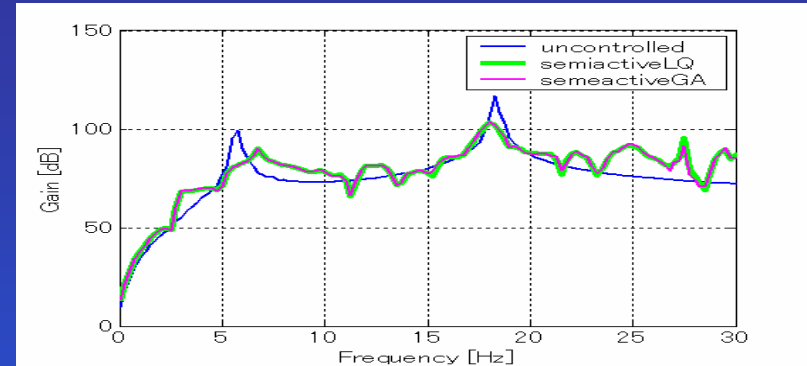
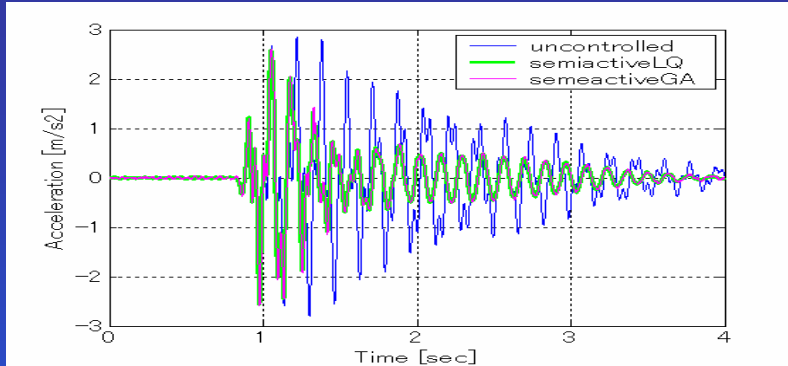
$$F_s = \begin{cases} F_{\max} & (u \times v > 0, |F_{\max}| < u) & \dots \text{Case 1} \\ u & (u \times v > 0, |F_{\min}| < u < |F_{\max}|) & \dots \text{Case 2} \\ F_{\min} & (u \times v > 0, |F_{\min}| > u) & \dots \text{Case 3} \\ F_{\min} & (u \times v \leq 0) & \dots \text{Case 4} \end{cases}$$

# Experimental Setup

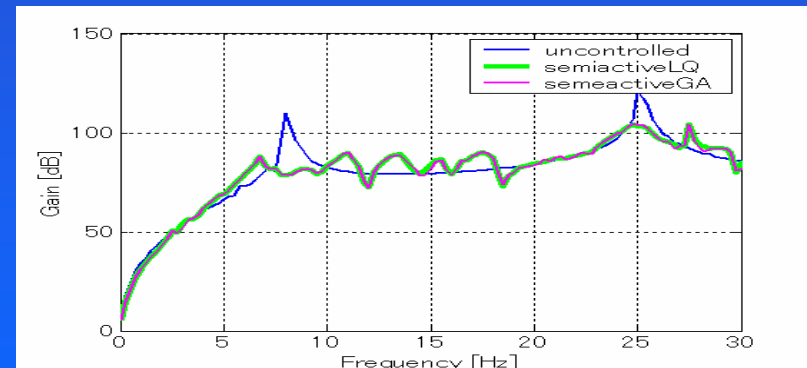
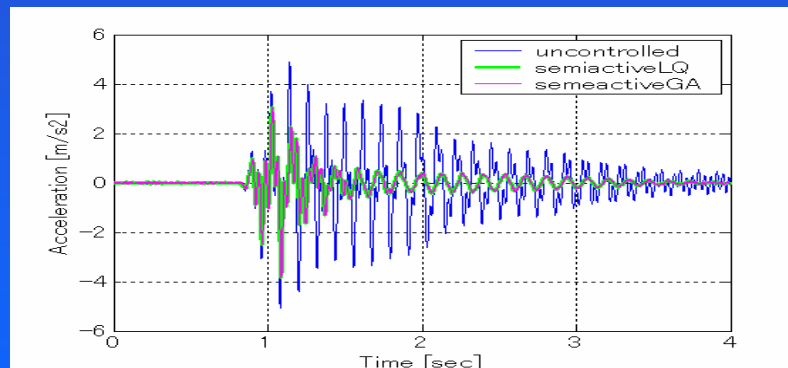


# Simulation Results

Top of structure 1



Top of structure 2

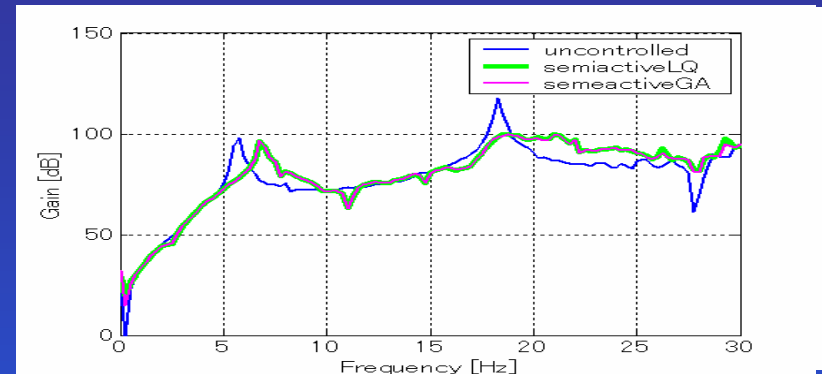
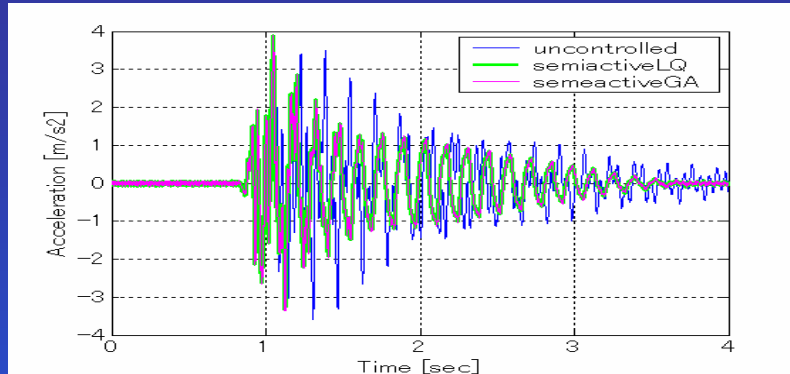


Acceleration time responses

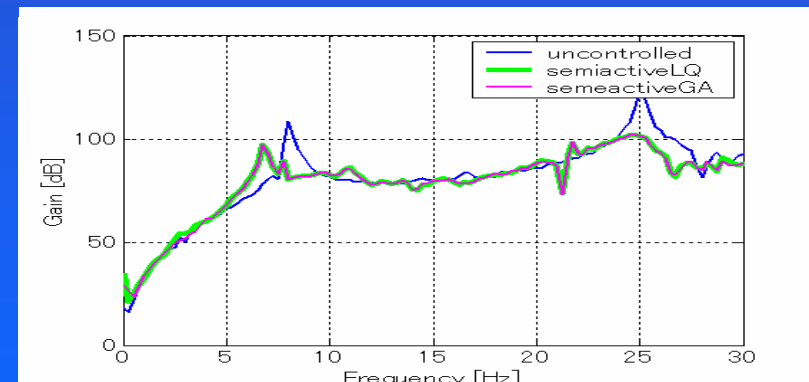
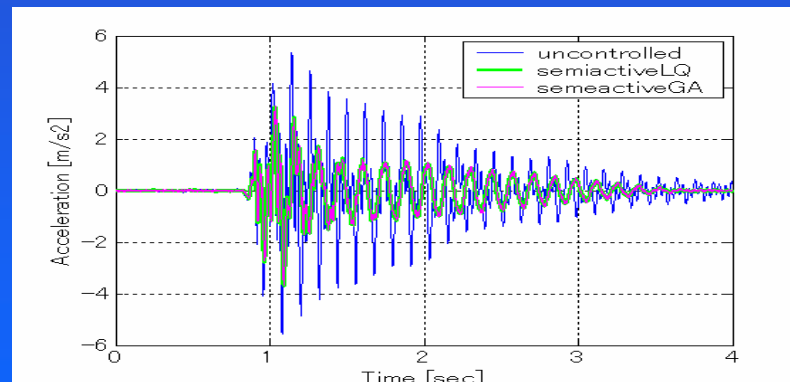
Frequency responses

# Experimental Results

Top of structure 1



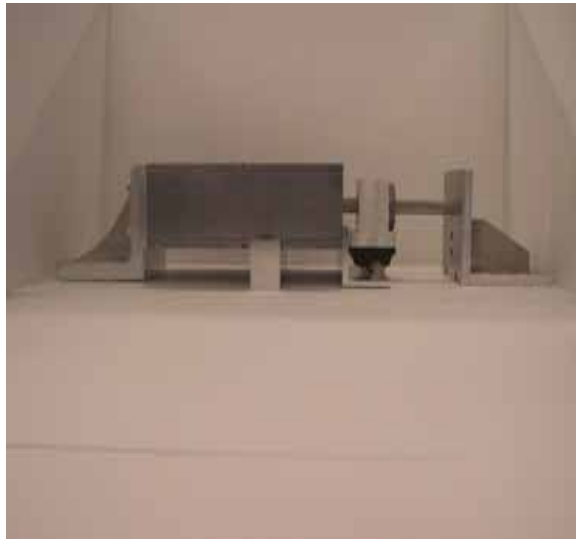
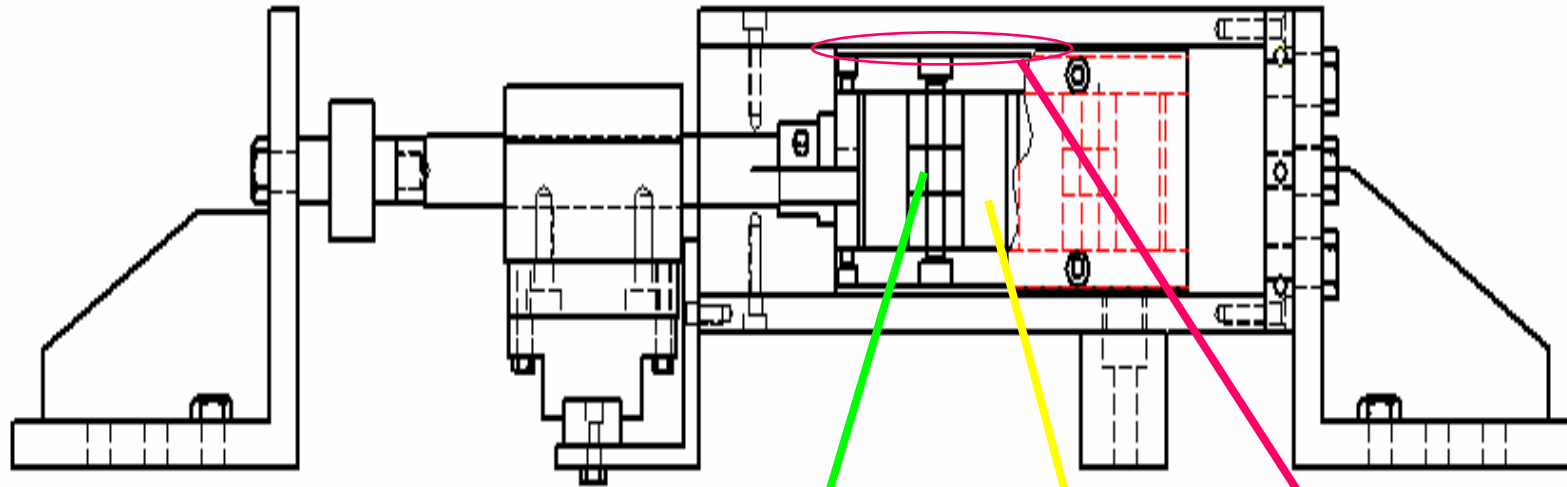
Top of structure 2



Acceleration time responses

Frequency responses

# Upgrade MR damper



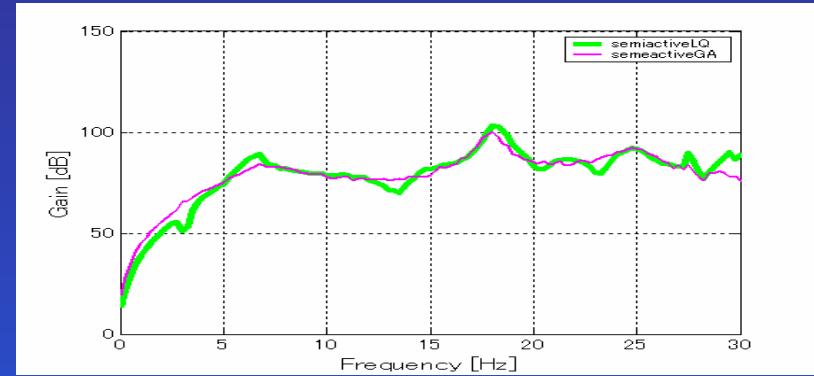
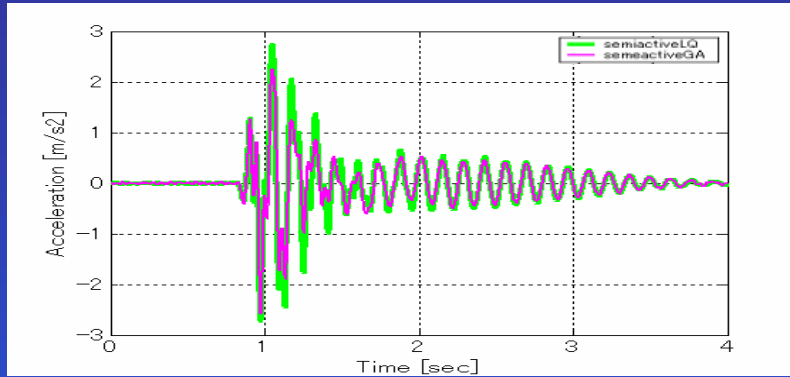
**Permanent  
magnet**

**Electric  
magnet**

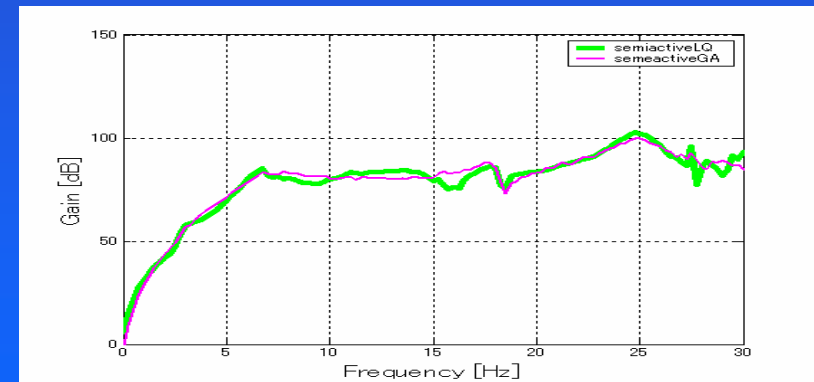
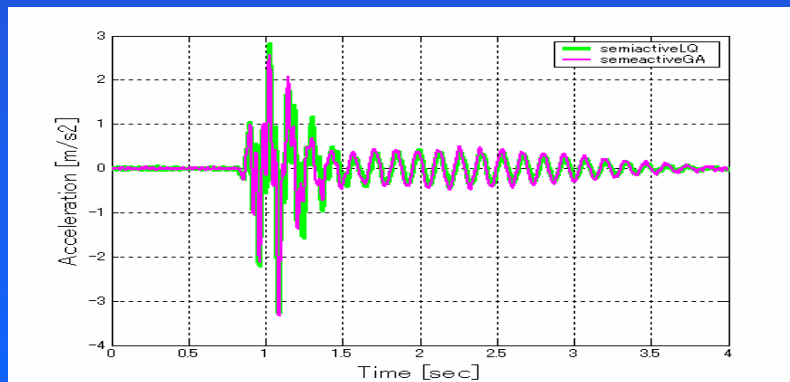
**MR fluid**

# Simulation Results With Upgrade Damper

Top of structure 1



Top of structure 2



Acceleration time responses

Frequency responses

## Conclusions

- 1 . A new sponge type MR damper mechanism was presented, and its capability was confirmed through experiments.
- 2 . The method using GA was effective for the nonlinear semi-active system better than using LQ theory.
- 3 . The control algorithm for semi-active system suppressed two vibration modes by one actuator.



*Thank you  
for your attention*

# Damping Devices

## Passive Systems

- Not adjusted from outside
- No energy is required

## Active Control Systems

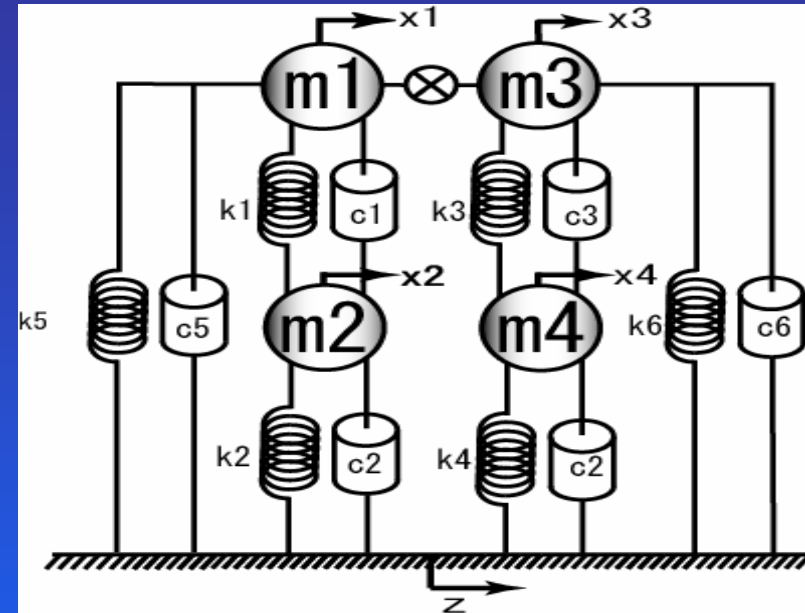
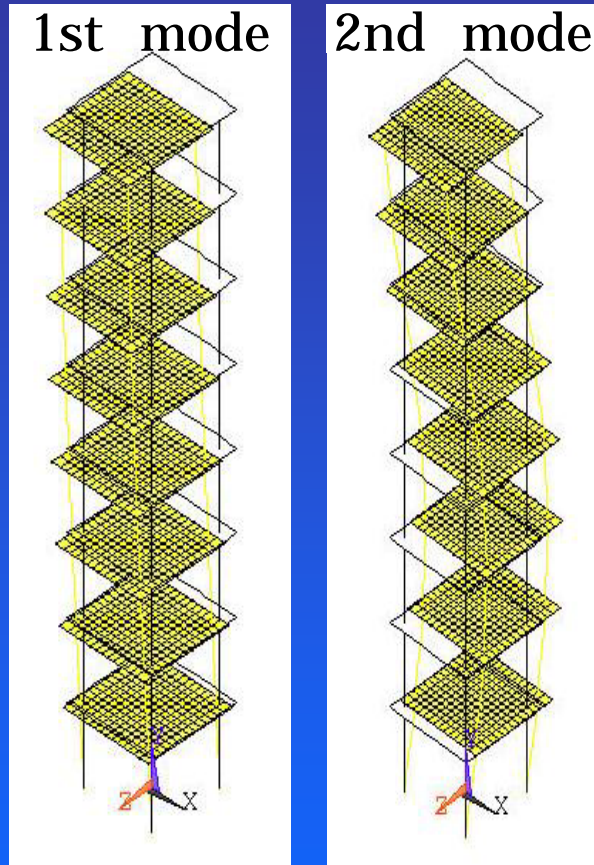
- Controllable
- Large energy is required

## Semi-active Systems

- Controllable
- Little energy is required

**A sponge type hybrid MR damper is used**

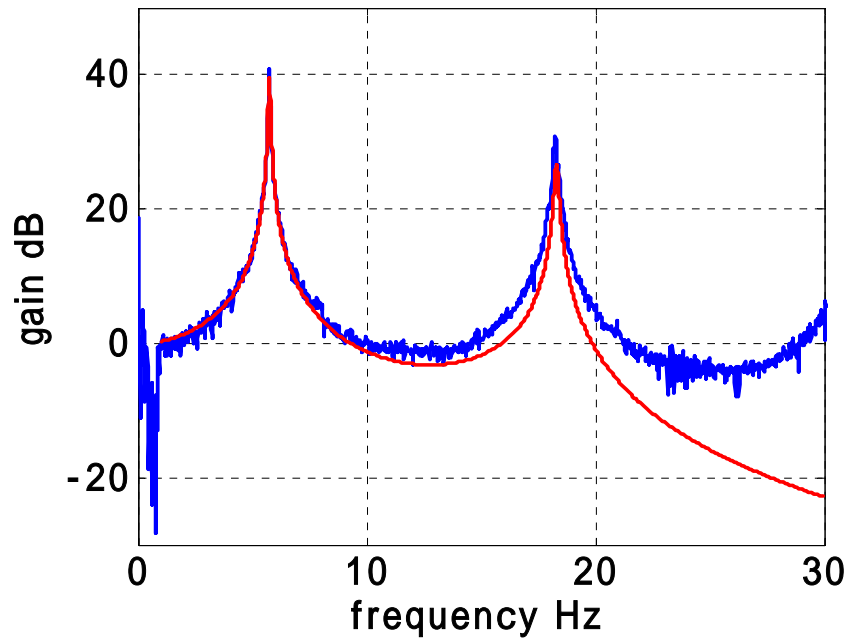
# Vibration Mode Shapes and Controlled Model



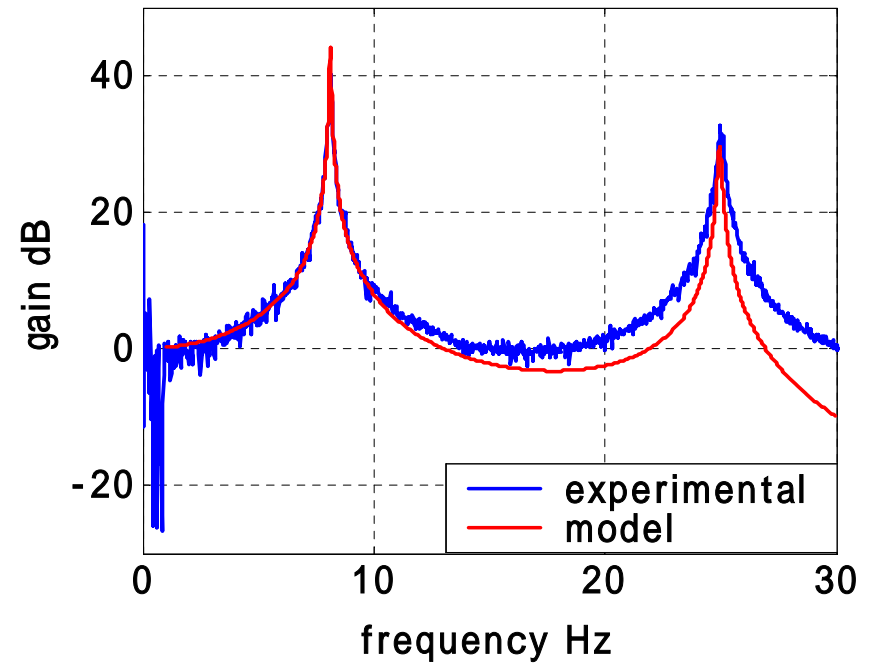
Structure1	5.55Hz	16.69Hz
Structure2	7.96Hz	23.90Hz

Mass[kg]	Stiffnes[N/m]	Damping[Ns/m]
m1=35.58	k1=1.89 × 10 <sup>5</sup>	c1=10
m2=43.61	k2=2.81 × 10 <sup>5</sup>	c2=47
	k11=-5.62 × 10 <sup>4</sup>	c11=4
m3=27.61	K3=3.08 × 10 <sup>5</sup>	c3=6
m4=34.75	k4=3.53 × 10 <sup>5</sup>	c4=60
	k22=-6.91 × 10 <sup>4</sup>	c22=10

# Verification of the Model



Top of Structure 1



Top of Structure 2

# MR Fluid

**MR fluid (Magneto-Rheological Fluid)**

... : **The fluid suspending magnetic particles in oil**

**In magnetic field, The magnetic particles form clusters.**

**These clusters become the flow resistance and increase the apparent viscosity of MR fluid.**

**This fluid can be controlled electrically using electric magnet.**



**by Lord co.**

# Genetic Algorithm (GA)

GA emulates Darwin's theory of evolution, and is the algorithm to search a broader region and to find an optimal solution.

• Semi-active control guarantees the stability, because its device is a kind of passive device.



The system is stable, even if the feedback control is used.

• The optimal solution may be in an instable area which is not able to be searched with LQ theory.



GA can search both stable and instable area.

• A semi-active system design method has not been established yet.

**Feedback gain  $K$  is designed by GA.**

# State Equation

$$\dot{X} = AX + BU + EW$$

$$X = [\dot{x}_1 \quad \dot{x}_2 \quad \dot{x}_3 \quad \dot{x}_4 \quad x_1 \quad x_2 \quad x_3 \quad x_4]^T$$

$$A = \begin{bmatrix} \frac{c_1+c_{11}}{m_1} & \frac{c_1}{m_1} & 0 & 0 & \frac{k_1+k_{11}}{m_1} & \frac{k_1}{m_1} & 0 & 0 \\ \frac{c_1}{m_2} & \frac{c_1+c_2}{m_2} & 0 & 0 & \frac{k_1}{m_2} & \frac{k_1+k_2}{m_2} & 0 & 0 \\ 0 & 0 & \frac{c_3+c_{22}}{m_3} & \frac{c_3}{m_3} & 0 & 0 & \frac{k_3+k_{22}}{m_3} & \frac{k_3}{m_3} \\ 0 & 0 & \frac{c_3}{m_4} & \frac{c_3+c_4}{m_4} & 0 & 0 & \frac{k_3}{m_4} & \frac{k_3+k_4}{m_4} \\ 1 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 & 0 & 0 \end{bmatrix}$$

$$E = \begin{bmatrix} \frac{c_{11}}{m_1} & \frac{c_2}{m_2} & \frac{c_{22}}{m_3} & \frac{c_4}{m_4} & 0 & 0 & 0 & 0 \\ \frac{k_{11}}{m_1} & \frac{k_2}{m_2} & \frac{k_{22}}{m_3} & \frac{k_4}{m_4} & 0 & 0 & 0 & 0 \end{bmatrix}^T$$

$$B = \begin{bmatrix} -\frac{1}{m_1} & 0 & \frac{1}{m_3} & 0 & 0 & 0 & 0 & 0 \end{bmatrix}^T$$

$$W = [\dot{w} \quad w]^T$$